

**Problem 1: Solution** The function  $y(x)$  determining the deflection satisfies the following differential equation

$$\frac{\partial^4 y}{\partial x^4} = \frac{1}{EI} qg,$$

which can be directly integrated to yield

$$y(x) = \frac{qg}{24EI} x^4 + ax^3 + bx^2 + cx + d.$$

The constants  $a, b, c, d$  are found from boundary conditions at both fixed ends

$$y(0) = 0, \quad y'(0) = 0, \quad y(l) = 0, \quad y'(l) = 0.$$

Thus we find

$$y(x) = -\frac{qg}{24EI} x^2(l-x)^2$$

The maximal deflection is in the middle  $y_{max} = y(l/2)$  and has the value

$$y_{max} = \frac{qg}{384} \frac{l^4}{EI}.$$

The torque is found from the Euler-Bernoulli law

$$M = -EI \frac{d^2 y}{dx^2},$$

and takes the value of

$$M(0) = -\frac{1}{12} qgl^2,$$

The force is given by  $F = \frac{dM}{dx}$ , so at the ends we have

$$R = \frac{qgl}{2}.$$

**Notw** This problem can be solved also by starting from the Euler-Bernoulli law

$$\frac{d^2 y}{dx^2} = \frac{M(x)}{EI}$$

where  $M(x)$  is an external force moment acting on a cross-section of the rod at  $x$ . With the way the ends of the rod are fixed, there are reaction forces  $R$  and moments  $M_0$ , and by symmetry they must be of the same value at both ends. They contribute to the total force moment  $M(x)$ , just as the gravity of the rod does. The resulting total distribution of force moment has the form

$$M(x) = -\int_0^x qg\xi d\xi + Rx + M_0 = -\frac{1}{2} qgx^2 + Rx + M_0.$$

From this we find the deflection of the beam as

$$y(x) = \frac{1}{EI} \left[ -\frac{1}{24} qgx^4 + \frac{1}{6} Rx^3 + \frac{1}{2} M_0 x^2 + Cx + C_0 \right].$$

The constants  $R, M_0, C, C_0$  are determined from the terminal boundary conditions:

$$y(0) = 0, \quad y'(0) = 0, \quad y(l) = 0, \quad y'(l) = 0.$$

We thus find

$$y(x) = -\frac{qg}{24EI} x^2(l-x)^2, \quad y_{max} = \frac{qg}{384} \frac{l^4}{EI}$$

And the unknown force and moment are:

$$R = \frac{1}{2} qgl, \quad M_0 = -\frac{1}{12} qgl^2.$$